

Reynolds number: a) A laminar separation bubble of limited spanwise extent first appears in the middle of the wing, with attached laminar flow on either side. b) Onset of transition in the wing boundary layer causes a local collapse of separation in the middle, while laminar separation persists outboard, producing a characteristic "two-lobed" separation pattern. c) The outboard lobes of laminar separation spread to the extremities of the hinge line, the inner (transitional or turbulent) flow remaining attached. The three examples illustrated in Fig. 4 have been observed in flow visualization studies reported in Ref. 1.

Conclusion

Trailing-edge flap-induced separation data on 75° delta wings have been examined to determine incipient separation characteristics. While the delta wing data for the turbulent boundary layer correlate with two-dimensional results, in the laminar and transitional cases a nearly parallel shift to higher flap angles for incipient separation is found. More data are needed (particularly for a range of leading-edge sweep angle) to establish properly the distinctive features of incipient separation on delta wings highlighted in this Note.

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Postbuckling Analysis of Crossply Laminated Plates

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Nomenclature

a	= plate length in x direction
b	= plate width in y direction
w	= deflection of a point on median surface of plate in direction normal to the undeformed plate
E_L, E_T	= Young's moduli of lamina in parallel to fibre and perpendicular to fiber directions, respectively
G_{LT}	= shear modulus of lamina in LT plane

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μ_{TL}, μ_{LT}	= Poisson's ratios; first subscript denotes lateral direction, second denotes load direction
E_f, E_m	= Young's moduli of fibre and matrix, respectively
m	= crossply ratio (ratio of the total thickness of odd layers to the total thickness of even layers)
n	= total number of layers
r	= a/b , aspect ratio of the plate
V_m	= volume content of matrix
P	= applied load
P_o	= buckling load
E	= E_L
F	= E_T/E_L
μ	= μ_{LT}
η	= G_{LT}/E_L
δ	= $\eta(F - \mu^2)/F$
C_{II}	= $EF/(N - \mu^2)$
A	= amplitude of buckling mode

Introduction

POSTBUCKLING analysis of orthotropic plates is presented in Ref. 1. Chan² presented postbuckling analysis for arbitrarily laminated plates using energy methods. In this Note, an approximate solution (one term solution) to postbuckling problem of unsymmetrically stacked crossply laminated plate simply supported all over the edges is presented. The analysis is based on governing equations derived in Ref. 3.

Analysis

The governing equations for crossply laminated plates are obtained by deleting the inertia term from the governing equations of Ref. 3. These are recorded as follows:

$$L_1 w - L_3 \phi - L_n(\phi, w) = 0 \quad (1a)$$

$$L_2 \phi + L_3 w - \frac{1}{2} L_n(w, w) = 0 \quad (1b)$$

where ϕ is the Airy Stress Function.

$$L_1 = D_{11}^* (\partial^4 / \partial x^4) + 2(D_{12}^* + 2D_{66}^*) (\partial^4 / \partial x^2 \partial y^2) + D_{22}^* (\partial^4 / \partial y^4)$$

$$L_2 = A_{22}^* (\partial^4 / \partial x^4) + (2A_{12}^* + A_{66}^*) (\partial^4 / \partial x^2 \partial y^2) + A_{11}^* (\partial^4 / \partial y^4)$$

$$-L_3 = B_{21}^* (\partial^4 / \partial x^4) + (B_{11}^* + B_{22}^*) (\partial^4 / \partial x^2 \partial y^2) + B_{12}^* (\partial^4 / \partial y^4)$$

$$L_n(\phi, w) = \phi_{,xx} w_{,yy} + \phi_{,yy} w_{,xx} - 2\phi_{,xy} w_{,xy}$$

A^* , B^* , and D^* are the matrices as defined in Ref. 4.

The plate is assumed to be simply supported at all the edges. The deflection function satisfying the geometric boundary conditions is assumed as follows:

$$w = A \sin(\pi x/a) \sin(\pi y/b) \quad (2)$$

It is not possible to satisfy the force boundary conditions by the previous function as the moment depends, not only upon the curvatures, but in-plane forces too, for unbalanced laminated plates. Hence, a modified galerkin's method⁵ wherein the residues at the edges are minimized, is applied here.

Inplane boundary conditions considered are as follows:

$$\int_0^b (\phi_{,yy})_{x=0,a} dy = -P; \quad \int_0^b (\phi_{,xy})_{x=0,a} dy = 0$$

$$\int_0^a (\phi_{,xx})_{y=0,b} dx = 0; \quad \int_0^a (\phi_{,xy})_{y=0,b} dx = 0 \quad (3)$$

Substituting expression (2) in Eq. (1b), and solving it, ϕ satisfying Eq. (3) is obtained as,

$$\phi = \frac{N}{D} h A \sin \frac{\pi x}{a} \sin \frac{\pi y}{b} + \frac{A^2 h^2}{2} r^2 \left[\frac{1}{A_{22}^*} \cos \frac{2\pi x}{a} + \frac{1}{r^4 A_{11}^*} \cos \frac{2\pi y}{b} \right] - \frac{P}{2b} y^2 \quad (4)$$

where

$$N = B_{21}^* \frac{\pi^4}{a^4} + (B_{11}^* + B_{22}^*) \frac{\pi^4}{a^2 b^2} + B_{12}^* \frac{\pi^4}{b^4}$$

$$D = A_{22}^* \frac{\pi^4}{a^4} + (2A_{12}^* + A_{66}^*) \frac{\pi^4}{a^2 b^2} + A_{11}^* \frac{\pi^4}{b^4}$$

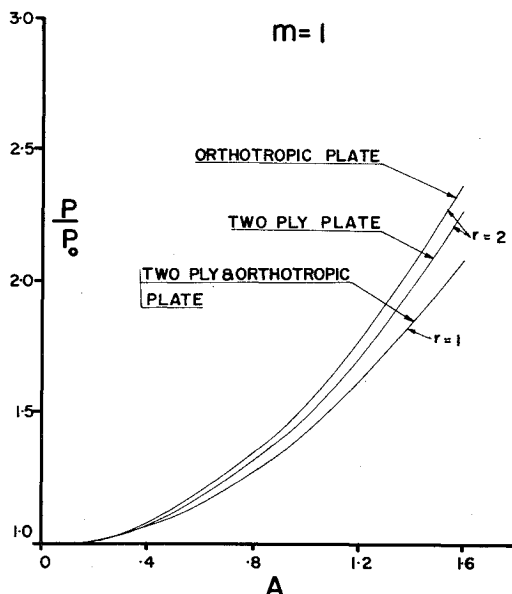


Fig. 1 Load-deflection curve for GFRP plate.

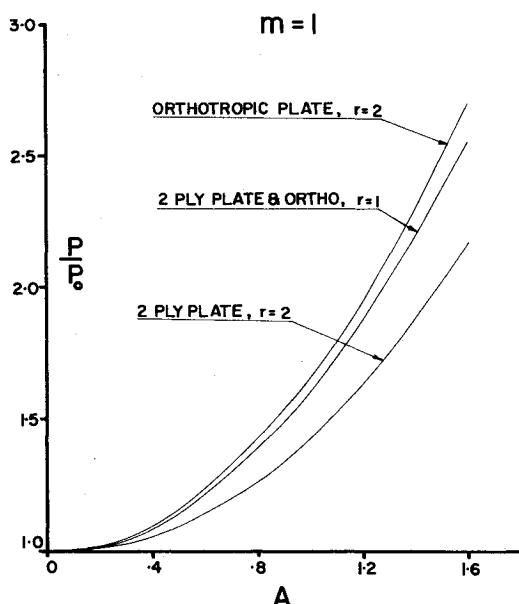


Fig. 2 Load-deflection curve for CFRP plate.

Substituting ϕ and w in Eq. (1a) and applying a modified Galerkin's technique, following algebraic equation in A is obtained.

$$(\alpha - \pi^2 (h/4a)P)A + \beta A^2 + \gamma A^3 = 0 \quad (5)$$

where

$$\alpha = (C_{11} h^4 / a^3) b \pi^4 [d_{11}^* + 2r^2 (d_{12}^* + \delta/6) + d_{22}^* r^4 + \{b_{12}^* (r^4 - 1) + (b_{22}^* - b_{11}^*) r^2\}^2 \times \{1/a_{22}^* + r^2 (2/a_{12}^* + 1/\delta) + r^4/a_{11}^*\}^{-1}] \quad (6)$$

$$\beta = 8/3 (C_{11} h^4 / a^3) b \pi^2 r^2 [b_{12}^* (a_{22}^* - a_{11}^*) - 4\{b_{12}^* (r^4 - 1) + (b_{22}^* - b_{11}^*) r^2\} \times \{1/a_{22}^* + r^2 (2/a_{12}^* + 1/\delta) + r^4/a_{11}^*\}^{-1}] \quad (7)$$

$$\gamma = (C_{11} h^4 / 16a^3) b \pi^4 (a_{11}^* + r^4 a_{22}^*) \quad (8)$$

The other parameters are as defined in Ref. 3.

Buckling load is determined from Eq. (5) by deleting A^2 and A^3 terms from it. Hence

$$P_0 = (4a/h\pi^2) \alpha \quad (9)$$

Knowing applied load P in the postbuckling range, Eq. (5) can be easily solved for A . Thus the unknown amplitude in the linear buckling is determined. Knowing the deflection, stresses can easily be computed. Thus, the state of stresses and displacement in the postbuckling range is known.

Conclusion

Load deflection curves are plotted using Eq. (5) for GFRP (Glass Fibre Reinforced Plastics) and CFRP (Carbon Fibre Reinforced Plastics) plates. Material constants of these are taken from Ref. 3. It is noted from Figs. 1 and 2 that for a square plate (GFRP or CFRP) the bending-inplane coupling does not affect the load deflection curve much. But for rectangular plates this coupling influences the load-deflection curve quite a bit. It is also noted that this effect is more for CFRP plates as compared to GFRP plates. This is easily explained since for the 2 ply configuration, the bending-inplane coupling is more pronounced for CFRP plates as compared to GFRP plates. Figures 1 and 2 also reveal that for the same load less deflection is predicted by orthotropic solutions as compared to the 2 ply solutions. Here orthotropic solutions are obtained by neglecting bending-inplane coupling. Thus, it is inferred that considering unbalanced laminates as orthotropic for this particular problem would predict non-conservative deflections.

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